8-14. A W14×34 wide flange beam is used to support a vertical force $P \sim N(80, 8^2)$ kip as shown in the figure. The beam is 20-ft long and pinned connected at both of its ends. The modulus of elasticity follows $E \sim N(29 \times 10^6, (2 \times 10^6)^2)$ psi. Determine the distribution of the critical buckling load. Also, determine the probability buckling. Assume that *E* and *P* are independent.



Solution:

Section roperties of the $W14 \times 34$ wide flange beam are

$$A = 10 \text{ in}^2$$
, $I_v = 23.3 \text{ in}^2$

Then, the critical axial buckling load is

$$P_{cr} = \frac{\pi^2 E I_y}{\left(KL\right)^2} = \frac{(3.14)^2 (23.3)}{\left(1 \times 20 \times 12\right)^2} E = (0.004)E; \qquad K = 1.$$

Since $E \sim N \left(29 \times 10^6, \left(2 \times 10^6 \right)^2 \right)$ psi, we have

$$\mu_{P_{cr}} = (0.004) \,\mu_E = 115.78 \,\mathrm{kip}$$

$$\sigma_{P_{cr}} = (0.004)\sigma_E = 7.98 \,\mathrm{kip}$$

Thus, the critical buckling load follows $P_{cr} \sim N(115.78, 7.98^2)$ kip. Ans.

Set $Y = P_{cr} - P$, then $Y \sim N(\mu_Y, \sigma_Y^2)$, where

$$\mu_{Y} = \mu_{P_{cr}} - \mu_{P} = 35.78 \text{ kip}$$
$$\sigma_{Y} = \sqrt{\sigma_{P_{cr}}^{2} + \sigma_{P}^{2}} = \sqrt{\sigma_{P_{cr}}^{2} + \mu_{P}^{2}} = 11.3 \text{ kip}$$

Thus, the probability of failure is

$$p_f = \Pr(Y < 0) = \Phi\left(\frac{-\mu_Y}{\sigma_Y}\right) = \Phi\left(-3.1655\right) = 7.7411 \times 10^{-4}$$
 Ans.