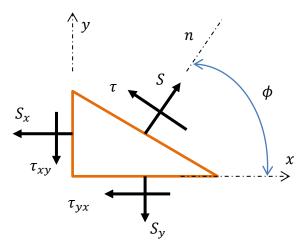
Exam 3

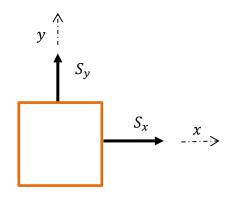
Please put your answers in the following table.

<u> </u>	1	2	2	4	F	(7	0	0	10
	1	Z	3	4	5	0	/	ð	9	10

1. An element is cut by an oblique plane with a normal *n* at an angle $\phi = 60^{\circ}$ counterclockwise from the *x* axis. If $S_x \sim N(60, 6^2)$ MPa, $S_y \sim N(50, 5^2)$ MPa, $\tau_{xy} \sim N(30, 3^2)$ MPa, what is the distribution of shear stress?

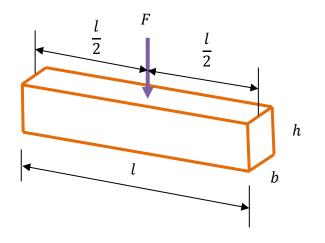


- A. $S \sim N(78.5, 7.8^2)$ MPaB. $S \sim N(78.5, 4.8^2)$ MPaC. $S \sim N(78.5, 5.6^2)$ MPaD. $S \sim N(78.5, 6.8^2)$ MPa
- 2. For the above problem, which of the following statements is not true about the first principle shear stress τ_1 ?
 - A. The smaller is the mean of τ_{xy} , the smaller is the mean of τ_1
 - C. The larger is the standard deviation of τ_{xy} , the larger is the standard deviation of τ_1
 - D. The larger is the standard deviation of σ_y , the smaller is the standard deviation of τ_1
 - D. τ_1 is not normally distributed
- 3. A stress element is subjected to two-dimensional stress as shown in the figure. The Poisson's ratio is $\nu = 0.3$ and the modulus of elasticity is E = 60 MPa. If $S_x \sim N(80, 8^2)$ MPa and $S_y \sim N(60, 6^2)$ MPa, determine the mean and standard deviation of the axial strain ϵ_x .



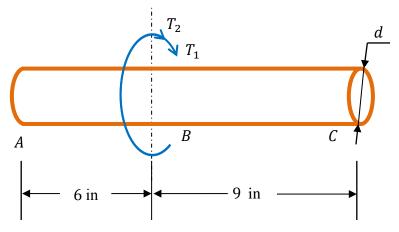
A. $\mu = 0.51$, $\sigma = 0.064$ B. $\mu = 0.51$, $\sigma = 0.051$ C. $\mu = 0.64$, $\sigma = 0.064$ D. $\mu = 0.64$, $\sigma = 0.051$

- 4. For problem 3, if the length of the element is l = 1 mm and the maximum allowable axial elongation is δ_a = 0.7 mm, determine the probability of failure.
 A. Φ(-2.95)
 B. Φ(-1.65)
 C. Φ(2.95)
 D. Φ(-2.68)
- 5. A concentrated load $F \sim N(6000, 600^2)$ N is applied to a beam with a rectangular cross-section as shown in the figure. The width and height of beam are b = 60 mm and h = 80 mm, respectively. And the length of the beam is l = 2 m. What is the distribution of the maximum bending stress?



A. $\tau_{\text{max}} \sim N(46.9, 22^2)$ MPa	B. $\tau_{\rm max} \sim N(46.9, 4.69^2)$ MPa
C. $\tau_{\text{max}} \sim N(64.9, 6.49^2)$ MPa	D. $\tau_{\text{max}} \sim N(64.9, 4.69^2)$ MPa

- 6. For the above problem, if the allowable bending stress is $S_a \sim N(70, 7^2)$ MPa, what is the reliability of the beam?
 - A. $\Phi(1.72)$ B. $\Phi(-2.75)$ C. $\Phi(2.75)$ D. $\Phi(-1.72)$
- 7. Two torques are applied to a round shaft as shown in the figure. The diameter of the shaft is d = 1 in. And the modulus of rigidity is $G = 11.5(10^6)$ psi If $T_1 \sim N(160, 16^2)$ lbf·in and $T_2 \sim N(120, 12^2)$ lbf·in, what is the distribution of the angle of twist at *B*?



A. $\theta \sim N(8.93(10^{-4}), (8.93(10^{-5}))^2)$ rad	B. $\theta \sim N(8.93(10^{-4}), (6.38(10^{-5}))^2)$ rad
C. $\theta \sim N(6.38(10^{-4}), (6.38(10^{-5}))^2)$ rad	D. $\theta \sim N(6.38(10^{-4}), (8.93(10^{-5}))^2)$ rad

- 8. For the above problem, if the allowable angle of twist is θ_a = 1(10⁻⁴), determine the probability of failure of the shaft.
 A. Φ(-2.5)
 B. Φ(2.5)
 C. Φ(-1.68)
 D. Φ(1.68)
- 9. A shaft is subjected to a torque $T \sim N(200, 20^2)$ N·m. If the shaft speed is 2500 rev/min, what is the distribution of the power that the shaft can transmit?

A. $\mu = 52.4 \text{ kW}, \sigma = 5.24 \text{ kW}$ B. $\mu = 5.24 \text{ kW}, \sigma = 0.524 \text{ kW}$ C. $\mu = 62.8 \text{ kW}, \sigma = 6.28 \text{ kW}$ D. $\mu = 6.28 \text{ kW}, \sigma = 0.628 \text{ kW}$

10. A force $F \sim N(60, 6^2)$ kN is applied to a truss as shown in the figure. The rod *BC* has a round cross-section with a dimeter of d = 0.1 m. If the yield stress of rod *BC* is $S_y = 20$ MPa, determine the reliability of rod *BC*.

